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BRUENINGHAUS HYDROMATIK GMBH

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**New claims**

1. Hydraulic assembly, situated in the housing (2) of which are a continuation bore (3) embodying a first pressure side (26), a threaded bore (1) embodying a second pressure side (27), and between the first pressure side (26) and the second pressure side (27), a nonreturn valve (5) having an external thread (11) which is formed on a first cylindrical portion (8) of a cylindrical valve housing (6) and can be screwed into the threaded bore (1),  
a passage duct (39) for a hydraulic fluid flow being formed, between a side wall of the threaded bore (1) and a first region of material removal (16) of the lateral surface (10) of the cylindrical valve housing (6), in a plurality of angular segments ( $\alpha_1$ ,  $\alpha_2$ ,  $\alpha_3$  and  $\alpha_4$ ) of the valve housing (6),  
**characterised**  
in that the first regions of material removal (16) are continued in a second cylindrical portion (9) adjoining the first cylindrical portion (8) and serve, with a plurality of second regions of material removal (17) situated between the first regions of material removal (16) exclusively in the second cylindrical portion (9), as engagements for a tool for screwing the nonreturn valve (5) into the threaded bore (1).

2. Hydraulic assembly according to Claim 1,  
**characterised**  
in that the cylindrical valve housing (6) comprises  
two, three or four equal-sized first regions of  
5 material removal (16) formed at equidistant angular  
intervals on the lateral surface (10) of the  
cylindrical valve housing (6).
3. Hydraulic assembly according to Claim 2,  
10 **characterised**  
in that in the second cylindrical portion (9) the two,  
three or four second regions of material removal (17),  
which are equal in size to the first regions of  
material removal (16) are constructed in the angular  
15 segments ( $\alpha 5$ ,  $\alpha 6$ ,  $\alpha 7$  and  $\alpha 8$ ) of the valve housing (6).
4. Hydraulic assembly according to Claim 3,  
**characterised**  
in that the first and second regions of material  
20 removal (16, 17) constitute levelled regions and form  
a square, hexagonal or octagonal profile for a tool  
for screwing the nonreturn valve (5) into the threaded  
bore (1).
- 25 5. Hydraulic assembly according to one of Claims 1 to 4,  
**characterised**  
in that the threaded bore (1) merges, at the level of  
the end, facing towards the first pressure side (26),  
of the valve housing (6) screwed fully into the  
30 threaded bore (1), via a transition (4) into a  
continuation bore (3), the diameter of which is  
designed smaller than the diameter of the threaded  
bore (1).

6. Hydraulic assembly according to Claim 5,  
**characterised**  
in that the transition (4) has a conical form.
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7. Hydraulic assembly according to Claim 5 or 6,  
**characterised**  
in that the hydraulic fluid flow between the valve  
housing (6) and the transition (4) between the  
10 threaded bore (1) and the continuation bore (3) is  
interrupted by the valve housing (6) pressing against  
the transition (4).
8. Hydraulic assembly according to one of Claims 1 to 7,  
15 **characterised**  
in that the nonreturn valve (5) contains a valve seat  
(21) which is formed by a conical transition (40) from  
a first portion (19) of smaller inside diameter to a  
second portion (20) of larger inside diameter of a  
20 cutout (18) of the hollow-cylindrical nonreturn valve  
(5).
9. Hydraulic assembly according to Claim 8,  
**characterised**  
25 in that the first portion (19) of the cutout (18)  
forms a first inflow opening (28) of the nonreturn  
valve (5).
10. Hydraulic assembly according to Claim 9,  
30 **characterised**  
in that the nonreturn valve (5) has a second opening  
(31) at the end of the valve housing (6) opposite the  
first inflow opening (28).

11. Hydraulic assembly according to Claim 10,  
**characterised**  
in that the second portion (20) of the cutout (18)  
5 contains a spherical valve body (22) which is pressed  
against the valve seat (21) by the spring force of a  
prestressed spring (25) likewise situated in the  
second portion (20) of the cutout (18) and the  
pressure difference between the pressure prevailing at  
10 the second opening (31) and the pressure prevailing at  
the first inflow opening (28).
12. Hydraulic assembly according to one of Claims 8 to 11,  
**characterised**  
15 in that the hollow-cylindrical nonreturn valve (5) has  
in the second portion (20) of the cutout (18) a  
plurality of through-openings (38) which are  
distributed in equidistant angular segments ( $\beta$ ) on a  
circular line which is concentric with the  
20 longitudinal axis (37) of the nonreturn valve (5) and  
lies on the inner lateral surface of the valve housing  
(6), these through-openings opening into a region (39)  
of the second pressure side (27) of the threaded bore  
(1), which region is situated on the side of the first  
25 cylindrical portion (8) facing towards the first  
pressure side (26).
13. Hydraulic assembly according to Claim 11,  
**characterised**  
30 in that the spring (25) is prestressed between a first  
and second spring plate (23, 24).

14. Hydraulic assembly according to Claim 13,  
**characterised**  
in that the first and second spring plate (23, 24)  
have the same geometry.
- 5 15. Hydraulic assembly according to Claim 13 or 14,  
**characterised**  
in that the spring force of the prestressed spring  
(25) is transmitted to the valve body (22) via the  
10 first spring plate (23).
16. Hydraulic assembly according to one of Claims 13 to  
15,  
**characterised**  
15 in that the second spring plate (24) is supported  
against a snap ring (34) guided in an annular groove  
at the inner lateral surface of the hollow-cylindrical  
valve housing (6).
- 20 17. Hydraulic assembly according to one of Claims 13 to  
16,  
**characterised**  
in that the first and second spring plate (23, 24)  
each has an inner bore (32) for supplying the pressure  
25 prevailing at the second opening (31) to the valve  
body (22).